

* Dimensional

Boundary layer equations

Schlichting & Gersten (2017), pp. 145-146

No-slip condition

Empirical observation:

- the velocity drops to 0 at the surface, i.e., the velocity is continuous across the solid-liquid boundary
- the molecules of the fluid interact with the molecules of the solid
- for large Re , u_x drops to 0 across a thin boundary layer

However! The potential flow is uniquely determined by $u_n = 0$ at the surface (flow tangency / no penetration).

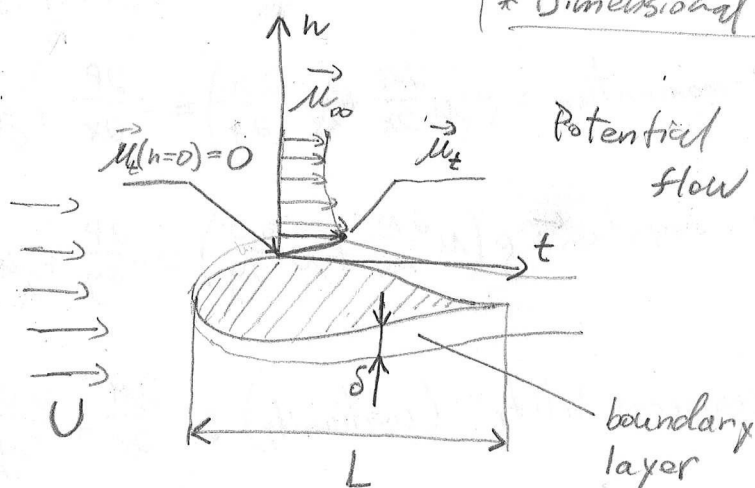
$\Rightarrow$ Not possible to prescribe both, $u_n = 0$ and $u_t = 0$, for the potential flow

$\Rightarrow$ Potential flow solution valid only outside the B.L.

Why does the potential-flow theory fail to describe the boundary layer (and the no-slip condition)?

$\rightarrow$ Remember under which assumptions we have derived the potential flow equations:

- Start from Navier-Stokes equations for a ^{steady} incompressible flow (conservation of momentum and mass/volume)



$$x\text{-momentum: } \rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = -\frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right)$$

$$y\text{-momentum: } \rho \left(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) = -\frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right)$$

$$\text{incompressibility (continuity): } \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

Order-of-magnitude analysis: Potential flow (S&G, p. 147)

- estimate the characteristic magnitudes of the variables:

$$u \sim \Delta u \sim U_\infty; v \sim \Delta v \sim U_\infty; \Delta x \sim L; \Delta y \sim L; \Delta p \sim ?$$

- estimate the magnitudes of all terms:

$$x\text{-momentum: } \rho u \frac{\partial u}{\partial x} \sim \rho v \frac{\partial u}{\partial y} \sim \frac{\rho U_\infty^2}{L}$$

$$\frac{\partial p}{\partial x} \sim \frac{\Delta p}{L}$$

$$\mu \frac{\partial^2 u}{\partial x^2} \sim \mu \frac{\partial^2 u}{\partial y^2} \sim \mu \frac{U_\infty}{L^2}$$

Observation: Pressure is essential for the kinematic BCs.

The pressure terms must compensate the convective term when the flow stagnates or deflects.

$$\Rightarrow \Delta p \sim \rho U_\infty^2 \Rightarrow \frac{\partial p}{\partial x} \sim \frac{\rho U_\infty^2}{L} \sim \rho \frac{D u}{D t}$$

Compare the magnitudes of the inertial and viscous terms:

$$\frac{\rho D\mathbf{u}/Dt}{\mu \nabla^2 \mathbf{u}} \sim \frac{\rho U_\infty^2}{L} / \frac{\mu U_\infty}{L^2} = \frac{\rho U_\infty L}{\mu} = Re \gg 1$$

$\Rightarrow$ viscous terms negligible

$\Rightarrow$ Euler equations $\Rightarrow$ Potential flow

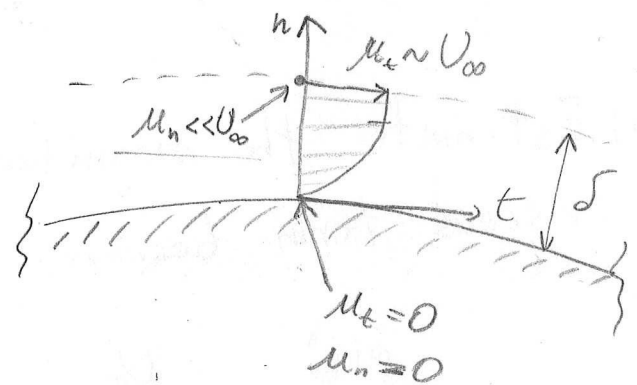
Boundary layer - different length-scales in wall-normal and wall-tangential directions!

$\rightarrow$ local coordinate system t - n

$$\Delta t \sim L, \text{ but } \Delta n \sim \delta !$$

$$u_t \sim \Delta u_t \sim U_\infty$$

$$u_n \sim \Delta u_n \ll U_\infty !$$



(small surface curvature)

$$\frac{\partial u_t}{\partial t} \sim \frac{U_\infty}{L} ; \quad \frac{\partial u_n}{\partial n} \sim \frac{\Delta u_n}{\delta} ;$$

Continuity equation: $\frac{\partial u_t}{\partial t} + \frac{\partial u_n}{\partial n} = 0 \Rightarrow \frac{\partial u_t}{\partial t} \sim \frac{\partial u_n}{\partial n}$

$$\frac{U_\infty}{L} \sim \frac{\Delta u_n}{\delta}$$

$$u_n \sim \Delta u_n \sim \frac{\delta U_\infty}{L} \ll U_\infty$$

Estimate magnitudes of different terms in the momentum equations $\rightarrow$ In class task $\leftarrow$ which terms are important?

t-momentum: $\rho \left(u_t \frac{\partial u_t}{\partial t} + u_n \frac{\partial u_t}{\partial n} \right) = - \frac{\partial p}{\partial t} + \mu \left(\frac{\partial^2 u_t}{\partial t^2} + \frac{\partial^2 u_t}{\partial n^2} \right)$

Compare convective terms:

1) $\rho u_t \frac{\partial u_t}{\partial t} \sim \rho \frac{U_\infty^2}{L}$; $\rho u_n \frac{\partial u_t}{\partial n} \sim \rho \frac{U_\infty}{L} \cdot \frac{U_\infty}{\delta} = \rho \frac{U_\infty^2}{L} \sim \rho u_t \frac{\partial u_t}{\partial t}$
 $\mu \frac{\partial^2 u_t}{\partial t^2} \sim \mu \frac{U_\infty}{L^2}$; $\mu \frac{\partial^2 u_t}{\partial n^2} \sim \mu \frac{U_\infty}{\delta^2}$

2) Compare the viscous terms:

$$\frac{\mu \partial^2 u_t / \partial n^2}{\mu \partial^2 u_t / \partial t^2} \sim \frac{L^2}{\delta^2} \gg 1 \Rightarrow \mu \frac{\partial^2 u_t}{\partial n^2} \gg \mu \frac{\partial^2 u_t}{\partial t^2}$$

3) Estimate the magnitude of $\frac{\delta}{L}$, such that the larger viscous term becomes comparable to the convective terms:

$$\frac{\rho U_\infty^2}{L} \sim \mu \frac{U_\infty}{\delta^2} \Rightarrow \frac{\delta^2}{L^2} \sim \frac{\mu}{\rho U_\infty L} = \frac{1}{Re}$$

$$\frac{\delta}{L} \sim \frac{1}{\sqrt{Re}} \quad (\text{S&G, p. 146, Eq. 6.1})$$

The pressure term is again:

$$\frac{\partial p}{\partial t} \sim \rho \frac{U_\infty^2}{L}$$

Side note: There exists a viscous sub-layer where $\frac{n}{L} \ll \frac{1}{\sqrt{Re}}$

such that only the viscous terms are important (4)

n-momentum:

$$\rho \mu_t \frac{\partial u_n}{\partial t} \sim \rho \frac{U_\infty^2 \delta}{L^2}; \quad \rho \mu \frac{\partial^2 u_n}{\partial t^2} \sim \rho \mu \frac{\delta U_\infty}{L^3}$$

$$\rho \mu_n \frac{\partial u_n}{\partial n} \sim \rho \frac{U_\infty^2 \delta}{L^2}; \quad \rho \mu \frac{\partial^2 u_n}{\partial n^2} \sim \rho \mu \frac{U_\infty}{L^2}$$

$$\rho \left(\mu_t \frac{\partial u_n}{\partial t} + \mu_n \frac{\partial u_n}{\partial n} \right) = -\frac{\partial p}{\partial n} + \rho \mu \left(\frac{\partial^2 u_n}{\partial t^2} + \frac{\partial^2 u_n}{\partial n^2} \right)$$

$$\rho \mu \frac{\partial^2 u_n}{\partial n^2} \gg \rho \mu \frac{\partial^2 u_n}{\partial t^2}$$

$$\frac{\rho D u_n / Dt}{\rho \mu \partial^2 u_n / \partial n^2} \sim \rho \frac{U_\infty^2 \delta}{L^2} / \rho \mu \frac{U_\infty}{L^2} = \frac{\rho U_\infty \delta}{\rho \mu} = Re \frac{\delta}{L} = \sqrt{Re} \gg 1$$

$$\frac{\partial p}{\partial n} \sim \frac{\rho U_\infty^2}{\delta}; \quad \frac{\rho D u_n / Dt}{\partial p / \partial n} \sim \rho \frac{U_\infty^2 \delta}{L^2} / \frac{\rho U_\infty^2}{\delta} = \frac{\delta^2}{L^2} \ll 1!$$

n-momentum:

$$\boxed{\frac{\partial p}{\partial n} = 0} \quad ? \quad (\text{assumption of small curvature})$$

$\Rightarrow$ pressure constant across the boundary layer

$\rightarrow$ determined by the potential flow solution

$\rightarrow$ classical attached laminar boundary layer does not affect the lift (at the leading order)

$\Rightarrow$ Computation of the flow in 2 steps:

1) Compute the potential flow with flow-tangency condition

2) Compute the boundary layer, using the ^{known} pressure from 1)

- B.L. equations are parabolic!

- enter pressure obtained from Bernoulli eq.

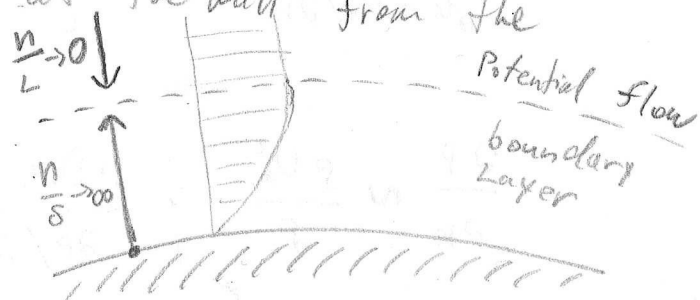
Boundary conditions for the boundary layer

- no-slip condition: $u_t = u_n = 0$ at $n=0$
- matching with the outer flow (continuous velocity):

$$u_t \rightarrow U_e \quad \text{as } n/\delta \rightarrow \infty \quad (\text{i.e., } n \gg \frac{L}{\sqrt{Re}})$$

where U_e is the tangential velocity at the wall from the potential flow solution:

$$U_e = \lim_{n/L \rightarrow 0} u_t$$



* "Van Dyke matching"

The boundary-layer solution provides the wall shear stress

$$\tau_w = \mu \left. \frac{\partial u_t}{\partial n} \right|_{n=0}$$

Note: In addition to friction drag, B.L. also creates "form drag"

$\Rightarrow$ skin-friction drag $\vec{F}_r = \oint \tau_w \vec{t} \, ds$

$\Rightarrow$ no stagnation pressure behind airfoil

* In the outer flow, the Bernoulli equation holds:

$$\frac{1}{2} \rho u^2 + p = \text{const.} = \frac{1}{2} \rho U_\infty^2 + p_\infty$$

$$p - p_\infty = \frac{1}{2} \rho (U_\infty^2 - u^2) \quad / \quad \nabla$$

$$\nabla p = -\frac{1}{2} \rho \nabla (u^2)$$

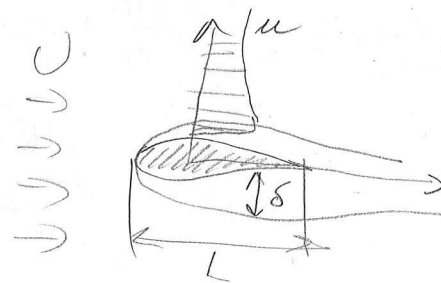
$$-\frac{\partial p}{\partial t} = \rho U_e \frac{\partial U_e}{\partial t}$$

Solution of B.L. eqs:
 $\Rightarrow$ Coordinate Transformation

easily obtained from Euler eq.

* In the boundary layer: $\frac{\partial p}{\partial n} = 0$

obtained from outer potential flow

Δ Dimensional

Recap: Close to the surface there is a boundary layer of thickness δ

- Inside the B.L., viscosity becomes comparable to inertia

$\Rightarrow$ B.L. cannot be described by potential flow
(the flow is not irrotational inside B.L.)

$\Rightarrow$ describing B.L. requires a new system of eqs.

- B.L. needed to describe the no-slip condition
- velocity drops to 0 at the surface

B.L. is the cause of drag and separation

Deriving B.L. eqs.: - different length scale in wall-normal direction

$$\Delta n \sim \delta$$

- new scale for wall-normal velocity:

$$\Delta u_n \sim ?$$

1) Continuity equation:

$$\frac{\partial u_t}{\partial t} + \frac{\partial u_n}{\partial n} = 0$$

$$\frac{U_\infty}{L} \sim \frac{\Delta u_n}{\delta}$$

$$\Delta u_n \sim \frac{\delta U_\infty}{L}$$

2) t-momentum:

$$\rho \left(u_t \frac{\partial u_t}{\partial t} + u_n \frac{\partial u_t}{\partial n} \right) = - \frac{\partial p}{\partial t} + \mu \left(\frac{\partial^2 u_t}{\partial t^2} + \frac{\partial^2 u_t}{\partial n^2} \right)$$

$$\frac{\rho U_\infty^2}{L}$$

$$\frac{\rho U_\infty^2}{L}$$

$$\frac{\rho U_\infty^2}{L}$$

$$\frac{\mu U_\infty}{L^2}$$

$$\frac{\mu U_\infty}{\delta^2}$$

$$\frac{\mu U_\infty}{\delta^2}$$

$$\frac{\rho U_\infty^2}{L}$$

$$\Rightarrow$$

$$\frac{\delta^2}{L^2} \sim \frac{\mu}{\rho U_\infty L}$$

Recap

(1)

n-momentum: $\rho \left(u_t \frac{\partial u_n}{\partial t} + u_n \frac{\partial u_n}{\partial n} \right) = - \frac{\partial p}{\partial n} + \mu \left(\frac{\partial^2 u_n}{\partial t^2} + \frac{\partial^2 u_n}{\partial n^2} \right)$

$$\frac{\rho U_\infty^2 \delta}{L^2} \quad \frac{\rho U_\infty^2 \delta}{L^2} \ll \frac{\rho U_\infty^2}{\delta} \quad \frac{\mu \delta U_\infty}{L^3} \ll \frac{\mu U_\infty}{L \delta^2}$$

$$\frac{\partial p}{\partial n} / \mu \frac{\partial^2 u_n}{\partial n^2} \sim \frac{\rho U_\infty^2 L \delta^2}{\mu \rho U_\infty} = \frac{\rho U_\infty L}{\mu} \cdot \delta \sim \frac{Re}{\sqrt{Re}} = \sqrt{Re}$$

$$\frac{\partial p}{\partial n} \gg \mu \frac{\partial^2 u_n}{\partial n^2}$$

$$\Rightarrow \left[\frac{\partial p}{\partial n} = 0 \right]$$

* Note: Bernoulli does not hold inside B.L.

Note: Surface curvature was neglected?

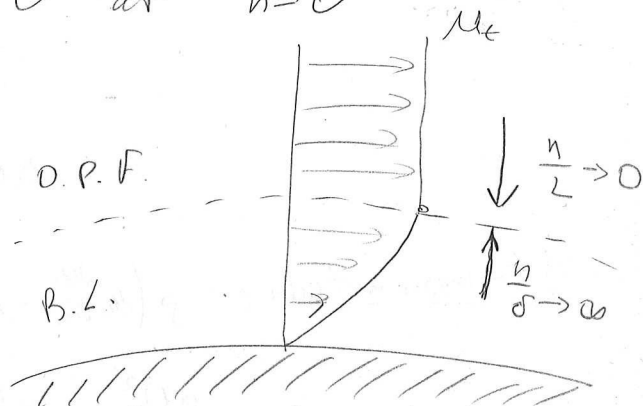
- Pressure across the boundary layer is constant (in n-direction)
- Pressure is continuous at the edge of B.L.
 $\Rightarrow$ "imposed" by the outer potential flow

Computation workflow: (Bernoulli equation)
 1) Compute pressure from outer pot. flow
 2) Solve B.L. equations with fixed pressure

Boundary conditions: - no-slip: $u_t = u_n = 0$ at $n=0$

- matching with the outer flow

$$\lim_{\frac{n}{\delta} \rightarrow \infty} u_t^{(B.L.)} = \lim_{\frac{n}{L} \rightarrow 0} u_t^{(P.F.)}$$



B.L. eq: $\rho \left(u_t \frac{\partial u_t}{\partial t} + u_n \frac{\partial u_t}{\partial n} \right) = \left(- \frac{\partial p}{\partial t} \right) + \mu \frac{\partial^2 u_t}{\partial n^2}$

tangential velocity at the wall from O.P.F.

From momentum eq. in O.P.F.

$$P - P_\infty = \frac{1}{2} \rho (u_\infty^2 - u^2) \Rightarrow - \frac{\partial p}{\partial t} = \rho u_e \frac{\partial u_e}{\partial t} ; \quad \text{Recap (2)}$$

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Thus, the boundary layer equations can be written as follows in dimensional form

$$t\text{-momentum: } \rho \left(u_e \frac{\partial u_e}{\partial t} + u_n \frac{\partial u_e}{\partial n} \right) = \rho u_e \frac{\partial u_e}{\partial t} + \mu \frac{\partial^2 u_e}{\partial n^2}$$

$$\text{continuity: } \frac{\partial u_e}{\partial t} + \frac{\partial u_n}{\partial n} = 0 \quad (\text{incompressible}) \quad (\text{S&G, p. 153})$$

Remarks: • B.L. eqs. are parabolic, that is, they describe an initial value problem in t -direction. Solution can be obtained by integration in t -direction, starting from given initial conditions.
 $\Rightarrow$ downstream conditions have no upstream influence
 - The eqs. become invalid near separation points!

Change of variables

1) Dimensionless form in outer-flow scaling:

$$\tilde{u} = \frac{u}{u_\infty}, \quad \tilde{v} = \frac{v}{u_\infty}, \quad \tilde{U} = \frac{U}{u_\infty}, \quad \tilde{x} = \frac{x}{L}, \quad \tilde{y} = \frac{y}{L} \quad (\text{S&G, p. 147})$$

→ substitute into (7.4):

$$\frac{u_\infty^2}{L} \tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{u_\infty^2}{L} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = \frac{u_\infty^2}{L} \tilde{U} \frac{d\tilde{U}}{d\tilde{x}} + \frac{v u_\infty}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} \cdot \frac{L}{u_\infty^2}$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = \tilde{U} \frac{d\tilde{U}}{d\tilde{x}} + \frac{1}{Re} \frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2}$$

⚠ However, $\tilde{v} \sim \frac{\tilde{u}}{\sqrt{Re}}$ and $\tilde{y} \sim \frac{\tilde{x}}{\sqrt{Re}}$ inside b.l. ! ↓

Therefore, introduce a

1) Boundary-Layer Scaling: (S&G, pp. 148-149)

$$\bar{u} = \frac{u}{u_\infty}, \quad \bar{v} = \frac{v}{u_\infty} \sqrt{Re}, \quad \bar{U} = \frac{U}{u_\infty}, \quad \bar{x} = \frac{x}{L}, \quad \bar{y} = \frac{y}{L} \sqrt{Re} \quad (6.6)$$

→ substitute into (7.4):

$$\left| \bar{u} \frac{\partial \bar{u}}{\partial \bar{x}} + \bar{v} \frac{\partial \bar{u}}{\partial \bar{y}} = \bar{U} \frac{d\bar{U}}{d\bar{x}} + \frac{\partial^2 \bar{u}}{\partial \bar{y}^2} \right| \quad (6.14)$$

Note that the viscous term does not vanish in the boundary-layer scaling as $Re \rightarrow \infty$!

2) Stream function

• dimensional: $u = \frac{\partial \psi}{\partial y}$, $v = -\frac{\partial \psi}{\partial x}$ (SRG, pp. 154-155, Eq. 6.37)

$\Rightarrow$ continuity equation (7.5) is identically satisfied

• dimensionless, in boundary layer scaling:

$$\bar{\psi} = \frac{\psi}{L u_{\infty}} \sqrt{Re}, \quad \bar{u} = \frac{\partial \bar{\psi}}{\partial \bar{y}} = \frac{\sqrt{Re}}{L u_{\infty}} \cdot \frac{L}{\sqrt{Re}} \frac{\partial \psi}{\partial y} = \frac{u}{u_{\infty}}$$

$$\bar{v} = -\frac{\partial \bar{\psi}}{\partial \bar{x}} = \frac{\sqrt{Re}}{L u_{\infty}} \cdot L \cdot \left(-\frac{\partial \psi}{\partial x}\right) = \frac{v}{u_{\infty}} \sqrt{Re}$$

$\rightarrow$ substituting into Eq. (6.14) we obtain:

$$\left[\frac{\partial \bar{\psi}}{\partial \bar{y}} \frac{\partial^2 \bar{\psi}}{\partial \bar{x} \partial \bar{y}} - \frac{\partial \bar{\psi}}{\partial \bar{x}} \frac{\partial^2 \bar{\psi}}{\partial \bar{y}^2} = \bar{u} \frac{d\bar{u}}{d\bar{x}} + \frac{\partial^3 \bar{\psi}}{\partial \bar{y}^3} \right]$$

(For dimensional form, see Schlichting and Gersten, 2017, p. 155, Eq. 6.38)